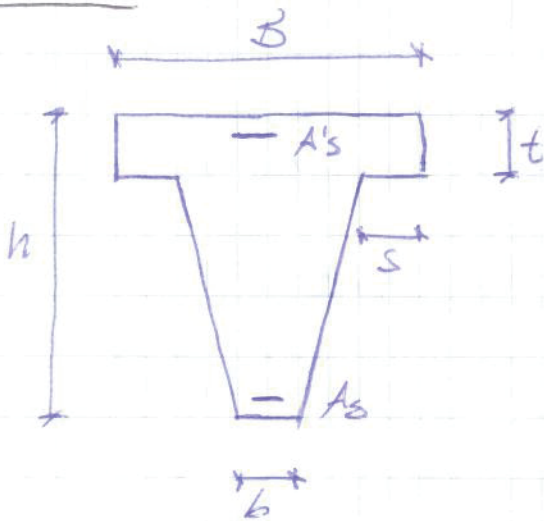


Ex. n. 3



$$b = 30 \text{ cm}$$

$$B = 46 \text{ cm}$$

$$h = 60 \text{ cm}$$

$$d' = 3 \text{ cm}$$

$$t = 20 \text{ cm}$$

$$s = 5 \text{ cm}$$

$$A_s = 18,84 \text{ cm}^2$$

$$A's = 6,03 \text{ cm}^2$$

$$M = 120 \text{ kNm}$$

$$C 20/25 \rightarrow f_{ck} = 20,75 \text{ MPa}$$

$$f_{cd} = \frac{20,75 \cdot 0,85}{1,5} = 11,75 \text{ MPa}$$

$$B450C \rightarrow f_{sd} = 391,3 \text{ MPa}$$

Verifica a flessione:

- Calcolo asse neutro

$H_p \rightarrow$ armature entrambe snervate e $\gamma_c < t$

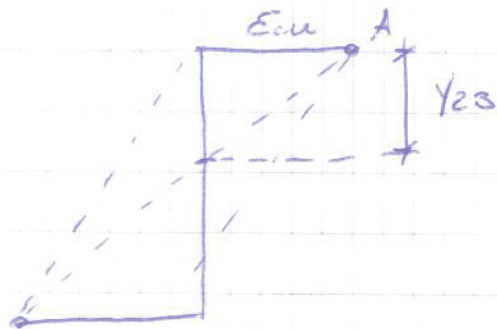
$$\varphi B f_{cd} \gamma_c + A's f_{sd} - A_s f_{sd} = 0$$

$$\gamma_c = \frac{(A_s - A's) f_{sd}}{\varphi B f_{cd}} = \frac{1281 \cdot 391,3}{0,8 \cdot 460 \cdot 11,75} = 115,9 \text{ mm}$$

$$\gamma_c = 11,59 \text{ cm} < t \text{ (verificato } \gamma_c < t)$$

- Si vanno a determinare le ϵ_s per verificare se sono snervate.

Si determina il polo di rotazione della sezione



$$\text{se } y_c < y_{23} \quad \text{polo B}$$

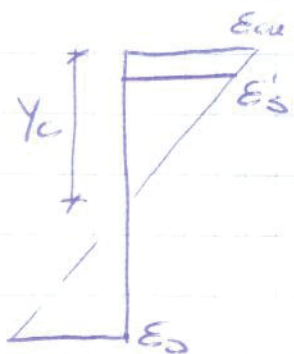
$$\text{se } y_c > y_{23} \quad \text{polo A}$$

$$B E_{ud} = 0,9 \cdot A_g = 6,75\%$$

$$E_{cu}: y_c = (E_{cu} + E_{ud}) \cdot d$$

$$y_{23} = \frac{E_{cu}}{E_{cu} + E_{ud}} \cdot d = \frac{0,0035}{0,0035 + 0,0675} \cdot 57 = 2,81 \text{ cm}$$

il polo di rotazione è A



$$E_{cu}: y_c = E_s' (y_c - d')$$

$$E_s' = \frac{E_{cu}}{y_c} \cdot (y_c - d') = 0,0026$$

$$E_s = \frac{E_{cu}}{y_c} (y_c - d) = -0,0137$$

$$\left[E_{sy} = \frac{f_{sd}}{E_s} = \frac{391,3}{210\,000} = 0,0019 \right]$$

$$|E_s'| > E_{sy} \quad \text{snervata}$$

$$|E_s| > E_{sy} \quad \text{snervata}$$

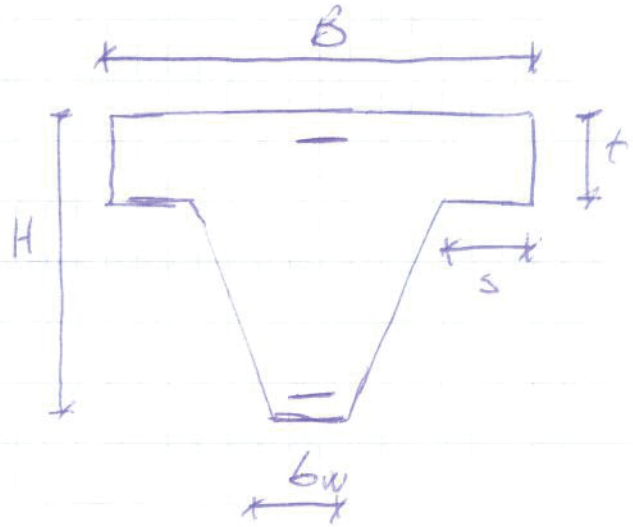
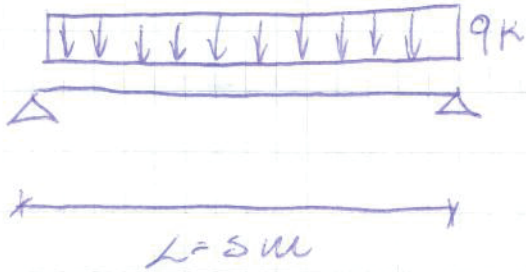
Le Hp. fatte sono soddisfatte $\rightarrow y_c = 11,59 \text{ cm}$.

$$\begin{aligned} M_{u0} &= \cancel{\phi B f'_{cd} y_c \left(\frac{h}{2} - y_c \right)} + \cancel{A_s f_{sd} \left(\frac{h}{2} - d' \right)} + \cancel{A_s f_{sd} \left(\frac{h}{2} - d' \right)} = \\ &= \cancel{0,8 \cdot 460 \cdot 11,75 \cdot 115,9 \left(300 - 0,4 \cdot 115,9 \right)} + \cancel{603 \cdot 391,3 \left(300 - 30 \right)} + \\ &+ \cancel{1884 \cdot 391,3 \left(300 - 30 \right)} = \end{aligned}$$

Si determina M_u rispetto all'armatura tesa per evitare di determinare la posizione del baricentro geometrico.

$$\begin{aligned} M_u &= \phi B f'_{cd} y_c (h - d' - y_c) + A_s f_{sd} (h - 2d') = \\ &= 0,8 \cdot 460 \cdot \overset{11,75}{\cancel{11,75}} \cdot 115,9 \left(600 - 30 - 0,4 \cdot 115,9 \right) + 603 \cdot 391,3 \left(600 - 60 \right) = \\ &= \underline{389,84 \text{ kNm}} > M_{sd} \quad \text{Verificata.} \end{aligned}$$

Esercizio 4

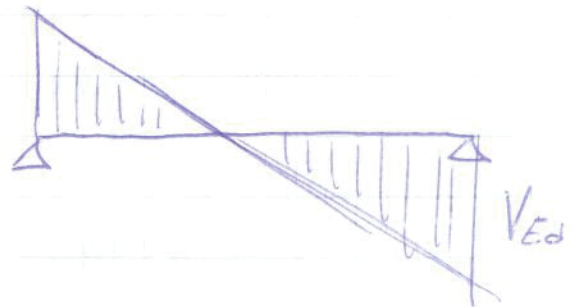


$$q_k = 18 \text{ kN/m}$$

$$q_k = \left\{ (B \cdot t) + \left[\frac{b_w + (B - 2s)}{2} \right] \cdot (H - t) \right\} \cdot 25 \text{ kN/m}^3 = 5,6 \text{ kN/m}$$

$$q_d = 1,3 q_k + 1,5 q_k = 34,28 \text{ kN/m}$$

$$T = \frac{q_d L}{2} = 85,7 \text{ kN}$$



Trave non Armata a taglio:

$$V_{Rd} = \left[\frac{0,18 \cdot K (100 \rho_L f_{ck})^{1/3}}{\gamma_c} \right] b_w \cdot d \geq V_{min} b_w d$$

$$K = 1 + \sqrt{\frac{200}{d}} = 1,592$$

$$V_{min} = 0,035 K^{3/2} f_{ck}^{1/2} = 0,035 \cdot 1,592^{3/2} \cdot 20,75^{1/2} = 0,32$$

$$\rho_L = \frac{A_s}{b_w d} = \frac{1884}{300 \cdot 570} = 0,011 \quad \left(A_s = \text{armatura inferiore perché essendo app-app. è sempre l'armatura tesa} \right)$$

$$V_{Rd} = \frac{0,18 \cdot 1,592 (100 \cdot 0,011 \cdot 20,75)^{1/3}}{1,5} \cdot 300 \cdot 570 \geq 0,32 \cdot 300 \cdot 570$$

$$V_{Rd} = 92,67 \text{ kN} \geq 57,75 \text{ kN} \Rightarrow V_{Rd} = 92,67 \text{ kN}$$

$$V_{Rd} > V_{Ed} (85,7 \text{ kN}) \quad \text{No Armatura a taglio}$$

↓
minimi normativi

~~Minimi normativi~~

in campata

$$\left\{ \begin{array}{l} A_{st} \geq 1,5 b = 1,5 \cdot 300 = 4,5 \text{ cm}^2/\text{m} \\ \rho_{st} \leq 0,8 d = 0,8 \cdot 570 = 456 \text{ mm} \\ 3 \text{ stette/m} \Rightarrow \rho_{st} \leq 300 \text{ mm} \end{array} \right.$$

$\Phi 8/20 \text{ cm}$
↓
 $A_{st} = 5 \text{ cm}^2/\text{m}$

sopra appoggi

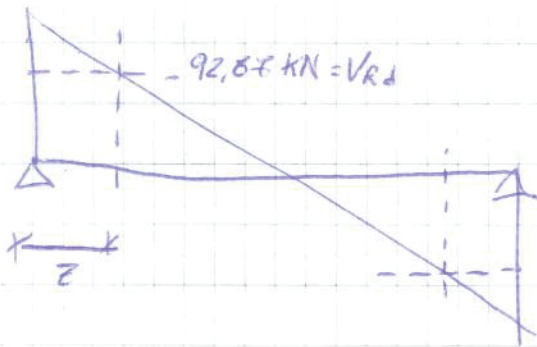
$$\left\{ \begin{array}{l} \rho_{st} \leq \frac{d}{4} = \frac{570}{4} = 14,25 \text{ cm} \\ 17,5 \text{ cm in } C\Delta^A \text{ e } 22,5 \text{ cm in } C\Delta^B \\ 6 \Phi_L \text{ o } 8 \Phi_L \text{ in } C\Delta^A \text{ o } C\Delta^B \\ 24 \Phi_{st} = 24 \cdot 0,8 = 19,2 \text{ cm} \end{array} \right.$$

$\Phi 8/10 \text{ cm}$

Per esercizio di progettazione ipotizziamo

$$V_{Ed} = 120 \text{ kN} > V_{Rd} \Rightarrow \text{Progettare a taglio}$$

120 kN



$$120 : \frac{L}{2} = 92,67 : x$$

$$x = \frac{92,67}{120} \cdot \frac{L}{2}$$

$$V_{Rd} = 0,9 b_w \cdot d \cdot f_{cd} \frac{(\cot \theta)}{1 + \cot^2 \theta} = V_{Ed}$$

$$\Downarrow \quad 0,9 \cdot 300 \cdot 570 \cdot 11,75 \frac{\cot \theta}{1 + \cot^2 \theta} = 120,00 \text{ kN}$$

$$\Downarrow \quad \cot \theta = \frac{15,00}{18,14} \quad \left. \begin{array}{l} 1 \leq \cot \theta \leq 2,5 \end{array} \right\} \Rightarrow \cot \theta = 2,5$$

$$\theta = 21,8^\circ$$

$$V_{Rd} = 0,9 d \frac{A_{st}}{s} f_{sd} (\cot \theta) = V_{Ed}$$

$$\Downarrow \quad A_{st} = \phi 8 \text{ zbr} \Rightarrow 1,00 \text{ cm}^2$$

$$s = \frac{0,9 d A_{st} f_{sd} \cot \theta}{V_{Rd}} = 41,82 \text{ cm}$$

Si armerà tenendo conto di minimo normativi.