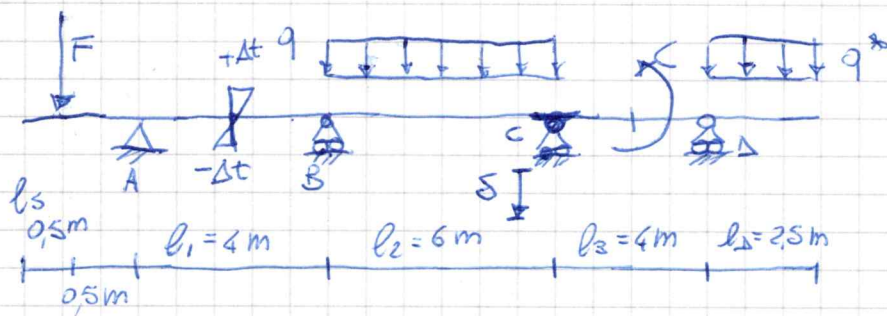


~~Trova continua~~

1

Trova continua



$$F = 120 \text{ kN}$$

$$C = 10 \text{ kNm}$$

$$q = 20 \text{ kN/m}$$

$$q^* = 16 \text{ kN/m}$$

$$\Delta t = 8.46 \text{ } ^\circ\text{C}$$

$$h = 200 \text{ mm}$$

$$I = 3600 \text{ cm}^4$$

$$E = 21 \cdot 10^5 \text{ N/mm}^2$$

$$\alpha = 12 \cdot 10^{-6} \text{ } ^\circ\text{C}^{-1}$$

$$\delta = 5 \cdot 10^{-4} \text{ m}$$

Non vi sono forze orizzontali, quindi il problema estensionale è
disaccoppiabile dal problema flessionale, e in questo caso la propria
soluzione $\epsilon_{long} = 0$

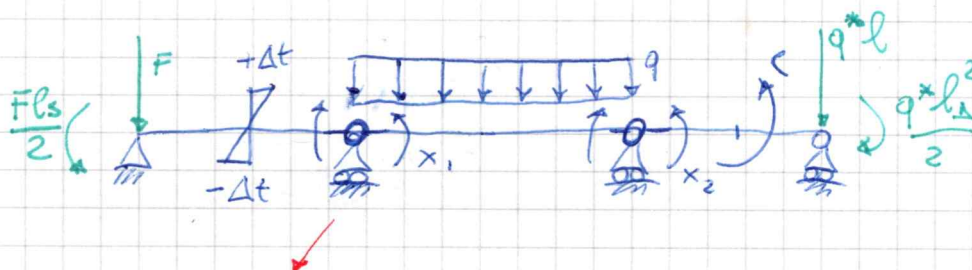
$$2t = 2$$

$$s = 4 \Rightarrow l = 2$$

Con il problema elastostatico risolvere la struttura è troppo lungo,
quindi si usa il metodo delle forze.

Non è utile operare sui vincoli ~~esterni~~ esterni, quindi si opera
decolando gli incastri interni in modo da avere più tronchi
riconducibili a schemi già noti.

È anche possibile eliminare gli sbalzi.



in inseriscono cerniere interne

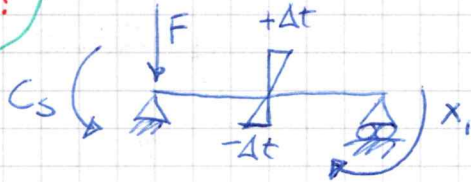
schemo elastico principale.

eq. di congruenza

(2)

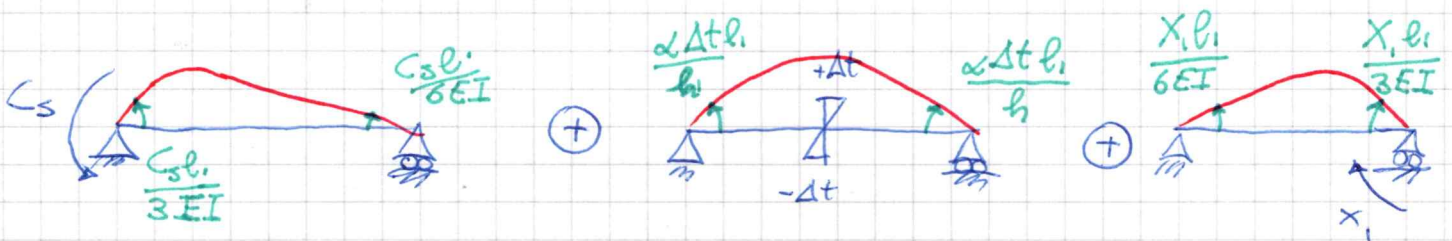
$$\begin{cases} \Delta\varphi_B = \varphi_{BA} - \varphi_{BC} = 0 \\ \Delta\varphi_C = \varphi_{CA} - \varphi_{CB} = 0 \end{cases}$$

$\varphi_{BA} = ?$



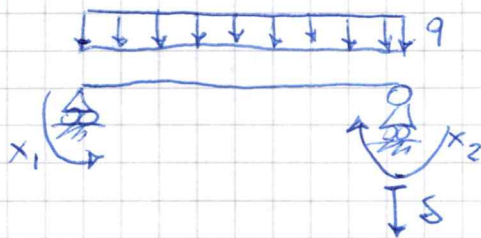
La F non dà rotazioni perché è assorbita dal vincolo

$$\varphi_{BA} = \varphi_{BA}^{(C_s)} + \varphi_{BA}^{(\Delta t)} + \varphi_{BA}^{(x_1)}$$

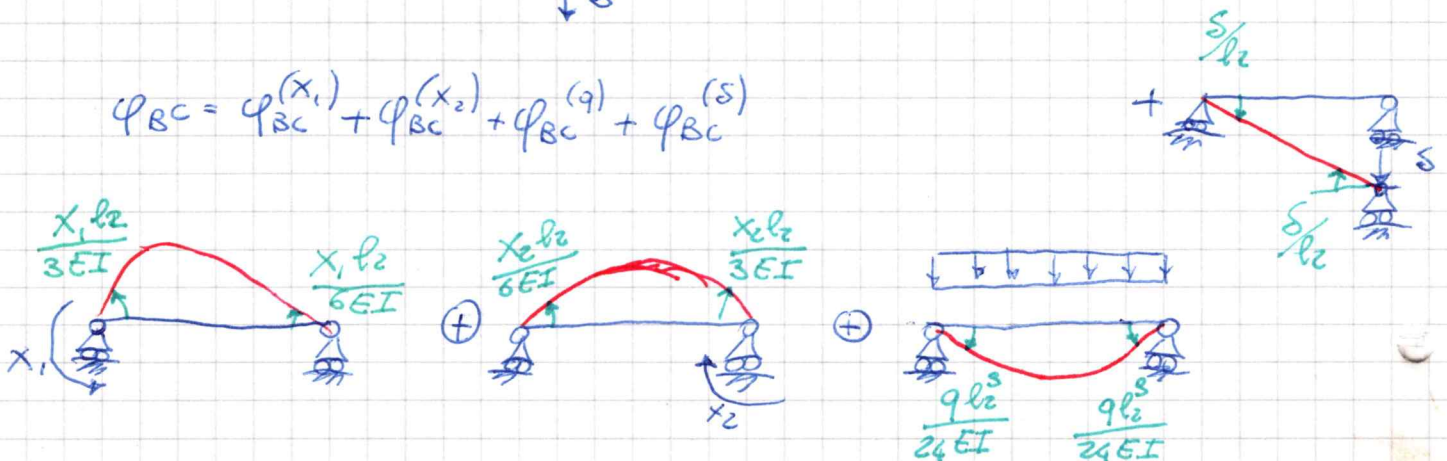


$$\varphi_{BA} = -\frac{C_s l_1}{6EI} - \frac{\alpha \Delta t l_1}{h} - \frac{x_1 l_1}{3EI}$$

φ_{BC}



$$\varphi_{BC} = \varphi_{BC}^{(x_1)} + \varphi_{BC}^{(x_2)} + \varphi_{BC}^{(q)} + \varphi_{BC}^{(s)}$$



$$\varphi_{BC} = \frac{X_1 l_2}{3EI} + \frac{X_2 l_2}{6EI} - \frac{ql^3}{24EI} - \frac{\delta}{l_2} \quad (3)$$

Quindi la prima eq. diventa.

$$\left\{ \varphi_{BA} - \varphi_{BC} = \frac{X_1 l_2}{3EI} + \frac{X_2 l_2}{6EI} - \frac{ql^3}{24EI} - \frac{\delta}{l_2} + \frac{C_s l_1}{6EI} + \frac{\alpha \Delta t l_1}{h} + \frac{X_1 l_1}{3EI} = 0 \right.$$

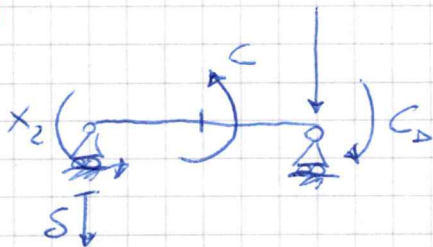
φ_{CB}

$$\varphi_{CB} = \varphi_{CB}^{(X_1)} + \varphi_{CB}^{(X_2)} + \varphi_{CB}^{(q)} + \varphi_{CB}^{(\delta)}$$

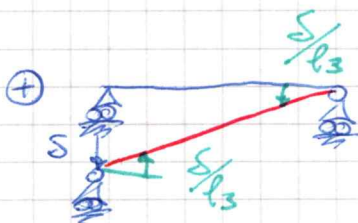
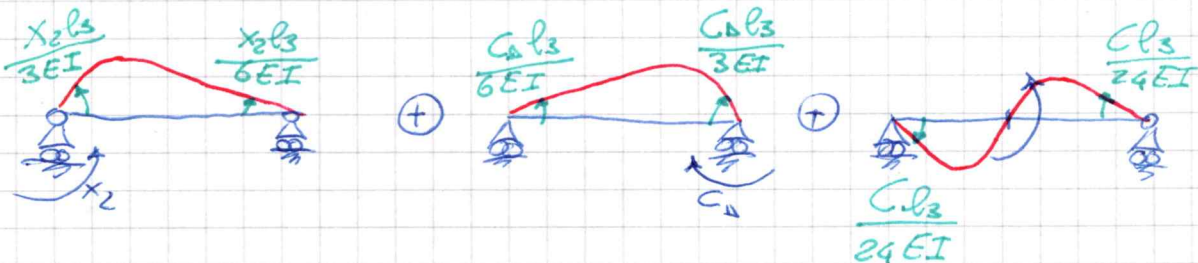
La trave è già diseguita per l'angolo φ_{BC}

$$\varphi_{CB} = -\frac{X_1 l_2}{6EI} - \frac{X_2 l_2}{3EI} + \frac{ql^3}{24EI} - \frac{\delta}{l_2}$$

φ_{CD}



$$\varphi_{CD} = \varphi_{CD}^{(X_2)} + \varphi_{CD}^{(C_D)} + \varphi_{CD}^{(\delta)} + \varphi_{CD}^{(q)}$$



$$\varphi_{CA} = \frac{X_2 l_3}{3EI} + \frac{C_\Delta l_3}{6EI} - \frac{C l_3}{24EI} + \frac{S}{l_3}$$

(4)

la seconda equazione diventa

$$\varphi_{CA} - \varphi_{CB} = \frac{X_2 l_3}{3EI} + \frac{C_\Delta l_3}{6EI} - \frac{C l_3}{24EI} + \frac{S}{l_3} + \frac{X_1 l_2}{6EI} + \frac{X_2 l_2}{3EI} - \frac{q l^3}{24EI} + \frac{S}{l_2} = 0$$

Le incognite iperstatiche sono X_1 e X_2

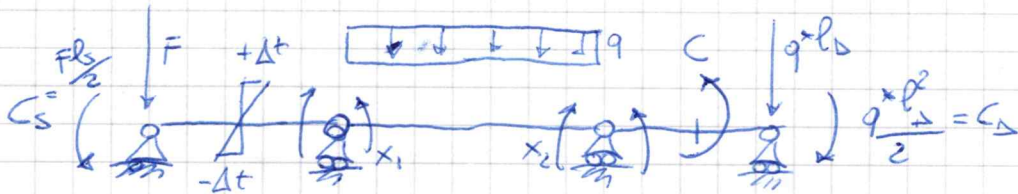
Risolvendo i valori numerici si ha il sistema lineare

$$\begin{cases} 4,41 \cdot 10^{-7} X_1 + 1,32 \cdot 10^{-7} X_2 = 1,68 \cdot 10^{-2} \\ 1,32 \cdot 10^{-7} X_1 + 4,41 \cdot 10^{-7} X_2 = 1,88 \cdot 10^{-2} \end{cases}$$

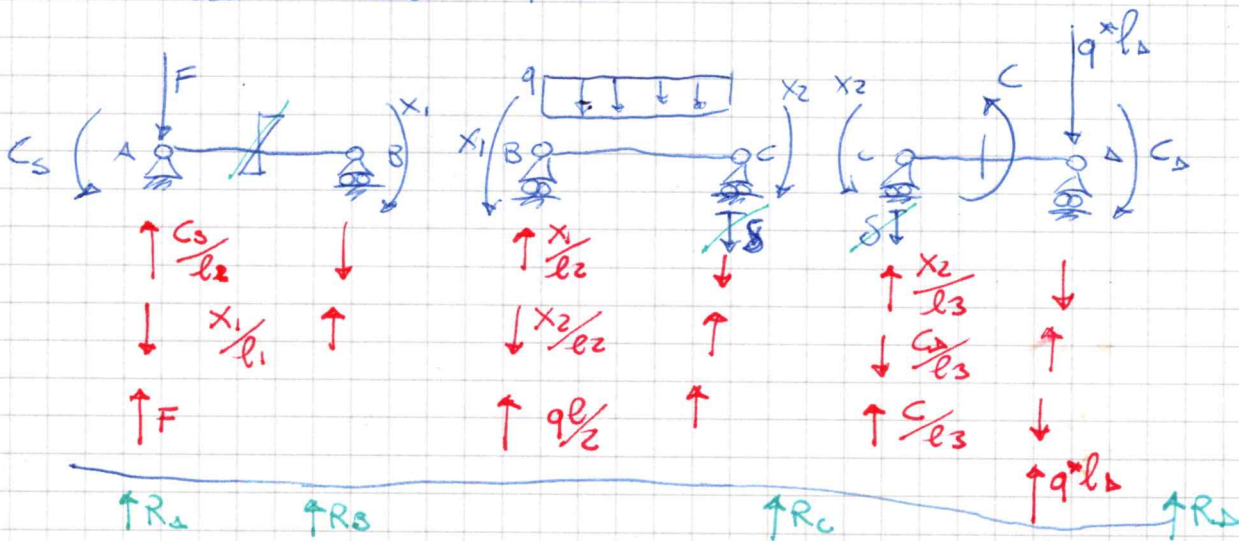
Se i calcoli sono giusti la matrice dei termini noti è simmetrica.

$$X_1 = 27,90 \text{ kNm}$$

$$X_2 = 34,21 \text{ kNm}$$



Determinare le reazioni vincolari



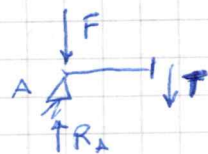
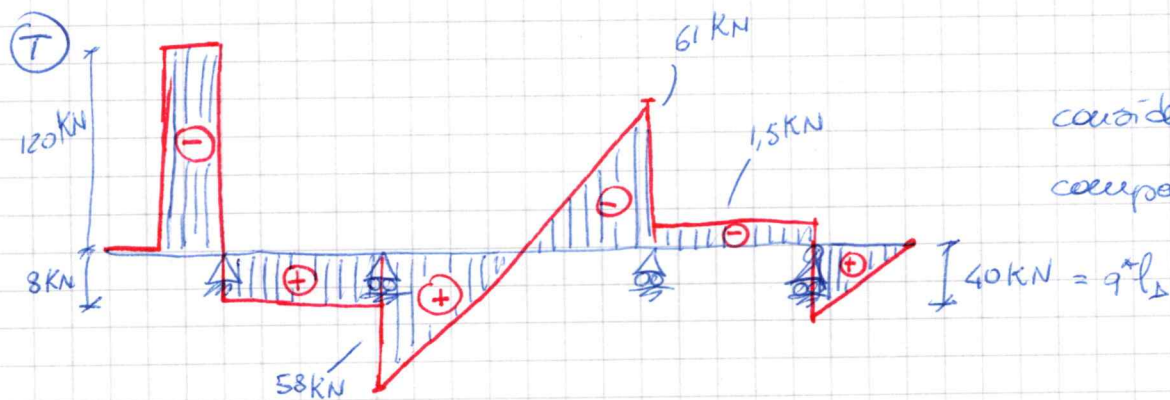
$$\underline{R_A} = \frac{C_5}{l_1} - \frac{X_1}{l_1} + F$$

$$\underline{R_B} = -\frac{C_5}{l_1} + \frac{X_1}{l_1} + \frac{X_1}{l_2} - \frac{X_2}{l_2} + \frac{ql}{2}$$

$$\underline{R_C} = -\frac{X_1}{l_2} + \frac{X_2}{l_2} + \frac{ql}{2} + \frac{X_2}{l_3} - \frac{C_4}{l_3} + \frac{C}{l_3}$$

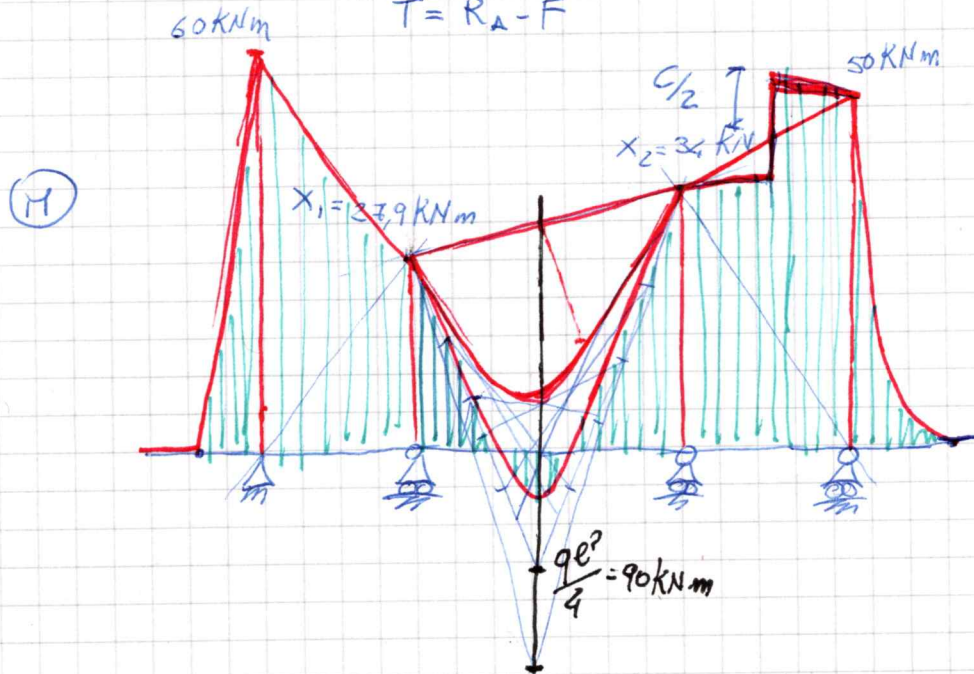
$$\underline{R_D} = -\frac{X_2}{l_3} + \frac{C_4}{l_3} - \frac{C}{l_3} + q^*l_3$$

Diagrammi (T. M. V)

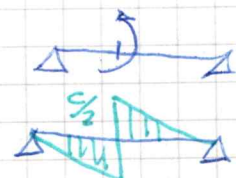


$$F + T - R_A = 0$$

$$\Rightarrow T = R_A - F$$

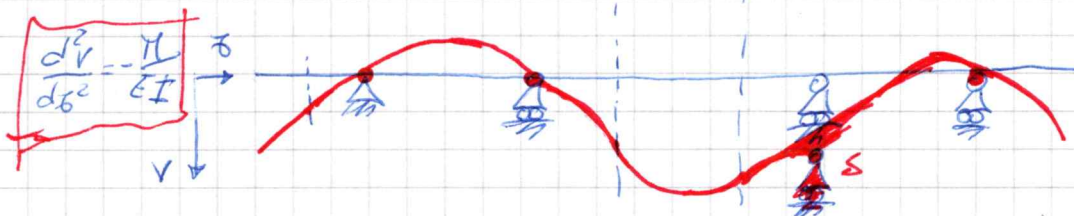


si considerano ora
il carico q e il
momento C riferendo
mi alla fondamentale
del momento.



punti di flesso presi dal momento

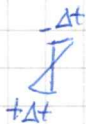
(V)



$$\theta + \theta^* = -\frac{d^2V}{dx^2}$$

$$-\frac{d^2V}{dx^2} = \frac{M}{EI} + \theta^*$$

$$\theta^* = \frac{2\alpha \Delta t}{h}$$



$$\left. \frac{d^2V}{dx^2} \right|_{x=x_A} = -\frac{M(x_A)}{EI} - \theta^* = \frac{60 \text{ kN}}{EI} + \frac{2\alpha \Delta t}{h} > 0$$

$$\left. \frac{d^2V}{dx^2} \right|_{x=x_B} = \frac{27}{EI} + \frac{2\alpha \Delta t}{h} > 0$$

la curvatura è ancora
positiva.